

Mecánica de Fluidos: Guía Técnica N°1

Problema N°1: Hallar $\frac{\nu}{\nu_0} = f(P, P_0, T, T_0)$

- Consideraciones previas: -) Considerar la ecuación de Sutherland
-) considerar ley de gases ideales
-) Estado de referencia inicial marcado con "0"; P_0, T_0, ν_0

Resolución:

Ec. Sutherland:
$$\frac{\mu}{\mu_0} = \left(\frac{T}{T_0}\right)^{\frac{3}{2}} \cdot \left(\frac{T_0 + S}{T + S}\right) \quad (1)$$

Ec. Gases Ideales:
$$P \cdot V = n \cdot R \cdot T \quad (2)$$

donde: μ : viscosidad cinemática
 S : constante de Sutherland
 T : temperatura

n : número de moles
 R : constante de los gases
 P : presión
 V : volumen

Definición:
$$\nu = \frac{\mu}{\rho} \quad (3)$$

ρ : densidad
 ν : viscosidad dinámica

Se busca:
$$\frac{\nu}{\nu_0} = \frac{\mu}{\rho} \cdot \frac{\rho_0}{\mu_0} \rightarrow \frac{\nu}{\nu_0} = \frac{\mu \cdot \rho_0}{\mu_0 \cdot \rho} \quad (4)$$

La densidad ρ se define:

$$\rho = \frac{m}{V} \quad \begin{array}{l} \rightarrow \text{masa} \\ \rightarrow \text{volumen} \end{array} \quad (5)$$

de (2):
$$P \cdot V = \frac{m}{M} \cdot R \cdot T$$

$$\frac{P \cdot V \cdot M}{R \cdot T} = m \quad ; \quad M: \text{ masa molar}$$

queda:
$$\rho = \frac{m}{V} = \frac{P \cdot M}{R \cdot T} \quad , \quad \text{donde } \frac{M}{R} = \text{constante}$$

$$\rho = P \cdot \frac{M}{R \cdot T} \quad (6) \quad \text{para } \rho_0 = P_0 \cdot \frac{M}{R \cdot T_0} \quad (7)$$

$$\frac{f_0}{f} = \frac{P_0 \cdot M}{R \cdot T_0} : \frac{P \cdot M}{R \cdot T} = \frac{P_0 \cdot \cancel{M} \cdot \cancel{R} \cdot T}{\cancel{P} \cdot \cancel{M} \cdot R \cdot T_0} \quad \boxed{\frac{f_0}{f} = \frac{P_0 \cdot T}{T_0 \cdot P}} \textcircled{3}$$

De (3), (1) e (4):

$$\frac{v}{v_0} = \frac{M \cdot f_0}{\mu_0 \cdot f} = \left(\frac{T}{T_0} \right)^{\frac{3}{2}} \left(\frac{T_0 + S}{T + S} \right) \cdot \frac{P_0}{P} \cdot \frac{T}{T_0}$$

$$\frac{v}{v_0} = \left(\frac{T}{T_0} \right)^{\frac{3}{2} + 1} \cdot \left(\frac{P_0}{P} \right) \cdot \left(\frac{T_0 + S}{T + S} \right)$$

$$\boxed{\frac{v}{v_0} = \left(\frac{T}{T_0} \right)^{\frac{5}{2}} \cdot \left(\frac{P_0}{P} \right) \cdot \left(\frac{T_0 + S}{T + S} \right)}$$

Otra forma:

Problema 1

$$\text{Haller } \left| \frac{V}{V_0} = f(P, P_0, T, T_0) \right| \text{ (1)}$$

Ec. Sutherland:

$$\left| \mu = \mu_0 \left(\frac{T}{T_0} \right)^{3/2} \cdot \left(\frac{1 + k \cdot T_0}{1 + kT} \right) \right| \text{ (2)}$$

Ec. gases ideales

$$\left| P \cdot V = n R T \right| \text{ (3)} \rightarrow \begin{cases} n R = \frac{P \cdot V}{T} \\ n R = \frac{P_0 \cdot V_0}{T_0} \end{cases}$$

$$V = \frac{M}{\rho} \rightarrow \frac{V}{V_0} = \frac{M}{\rho} ; \frac{\mu_0}{\rho_0} = \frac{M \cdot \rho_0}{\mu_0 \cdot \rho}$$

$$\left| \frac{V}{V_0} = \frac{M \cdot \rho_0}{\mu_0 \cdot \rho} \right| \text{ (4)}$$

$$\text{de (3); } \rho = \frac{m}{V}$$

$$\rho = \frac{R \cdot T}{P}$$

$$\frac{\rho_0}{\rho} = \frac{R \cdot T_0}{P_0} \cdot \frac{P}{R \cdot T} \quad \left| \frac{\rho_0}{\rho} = \frac{P \cdot T_0}{P_0 \cdot T} \right| \text{ (5)}$$

de (2), (5) e (4),

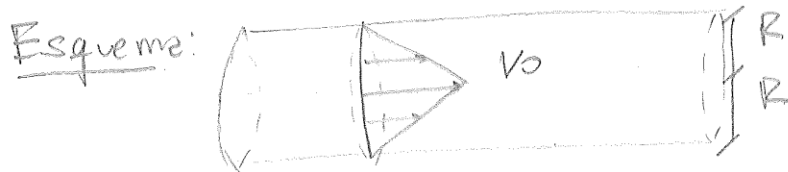
$$\frac{V}{V_0} = \left(\frac{T}{T_0} \right)^{3/2} \cdot \left(\frac{1 + k \cdot T_0}{1 + kT} \right) \cdot \frac{P}{P_0} \cdot \frac{T_0}{T}$$

$$\left| \frac{V}{V_0} = \left(\frac{T}{T_0} \right)^{1/2} \cdot \left(\frac{1 + k \cdot T_0}{1 + kT} \right) \cdot \frac{P}{P_0} \right|$$

Problema 2

a) Hallar V_{media} / V_{ese}

Datos: $V(r) = V_0 \left(1 - \frac{r}{R}\right)$ (1)



Consideraciones previas:) Se define Velocidad media V_m :

$$V_m = \frac{Q}{A} \quad (2)$$

Para hallar Q y A de (2) debemos integrar

) Tubo de sección circular.

Resolución: de (2): $dQ = V \cdot dA$ (3), para área circular:

$$A = \pi \cdot r^2 \rightarrow dA = r \, d\theta \, dr \quad (4)$$

de (4) y (3):

$$dQ = V \cdot dA \quad \text{integro: } A \begin{cases} \theta \in [0, 2\pi] \\ r \in [0, R] \end{cases}$$

$$Q = \int_0^Q dQ = \int_0^{2\pi} \int_0^R V(r) \cdot r \cdot dr \, d\theta \Rightarrow$$

$$\Rightarrow Q = \int_0^{2\pi} \int_0^R V_0 \left(1 - \frac{r}{R}\right) \cdot r \cdot dr \, d\theta \Rightarrow$$

$$\Rightarrow Q = V_0 \left[\int_0^{2\pi} \int_0^R \left(r - \frac{r^2}{R}\right) dr \, d\theta \right] \Rightarrow$$

$$\Rightarrow Q = V_0 \left[\int_0^{2\pi} \left(\frac{r^2}{2} \right) \Big|_0^R - \frac{r^3}{3R} \Big|_0^R \right] d\varphi \Rightarrow$$

$$\Rightarrow Q = V_0 \left[\int_0^{2\pi} \left(\frac{R^2}{2} - \frac{R^3}{3R} \right) d\varphi \right] \Rightarrow$$

$$\Rightarrow Q = V_0 \left[\int_0^{2\pi} \left(\frac{R^2}{2} - \frac{R^2}{3} \right) d\varphi \right] \Rightarrow$$

$$\Rightarrow Q = V_0 \int_0^{2\pi} \frac{R^2}{6} d\varphi \Rightarrow Q = V_0 \cdot \frac{2\pi \cdot R^2}{6}$$

$$\boxed{Q = V_0 \cdot R^2 \cdot \frac{\pi}{3}} \quad (3^\circ)$$

.) Calculo A. $A = \int_0^{2\pi} \int_0^R r dr d\varphi = \int_0^{2\pi} \left(\frac{r^2}{2} \right) \Big|_0^R d\varphi = R^2 \pi$

$$\boxed{A = \pi R^2} \quad (4^\circ)$$

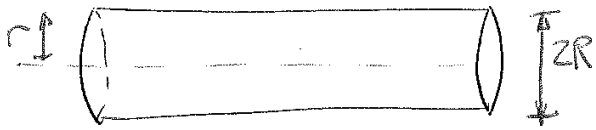
de (2), (3^o) y (4^o): $V_m = \frac{Q}{A} = \frac{V_0 \cdot R^2 \cdot \pi}{3 \cdot \pi \cdot R^2}$; $\boxed{V_m = \frac{V_0}{3}}$

3) $V_{\text{ave}} = ?$ $V_{\text{ave}} = V(r=0) = V_0 \left(1 - \frac{0}{R} \right) = V_0$ \therefore

$$V_m / V_{\text{ave}} = \frac{V_0}{3} : \frac{V_0}{1} \rightarrow \boxed{\frac{V_m}{V_{\text{ave}}} = \frac{1}{3}}$$

Problema 3

Perfil de velocidades: $u = U_0 \left(1 - \frac{r^6}{R^6} \right)$ ①



a) Caudal? $Q = u \cdot A$ ② ;

Q: caudal
A: área
u: velocidad

Para calcular, integramos la expresión ②:

$$dQ = u \cdot dA \quad ③$$

Tomamos elementos diferenciales de área:

$$A = \pi \cdot r^2 \rightarrow dA = 2\pi r dr \quad ④$$

Integramos ③ con ④ y ①

$$\int_0^Q dQ = \int_0^R u \cdot 2\pi r dr$$

$$Q = \int_0^R U_0 \left(1 - \frac{r^6}{R^6} \right) \cdot 2\pi r dr$$

$$Q = U_0 \cdot 2\pi \int_0^R \left(r - \frac{r^7}{R^6} \right) dr =$$

$$Q = U_0 \cdot 2\pi \left[\frac{R^2}{2} - \frac{1}{R^6} \cdot \frac{1}{8} \cdot R^8 \right]$$

$$Q = U_0 \cdot 2\pi \left[\frac{R^2}{2} - \frac{R^2}{8} \right], \quad Q = U_0 \cdot 2\pi \cdot R^2 \left[\frac{1}{2} - \frac{1}{8} \right]$$

$$Q = U_0 \cdot 2\pi \cdot R^2 \left[\frac{4-1}{8} \right]$$

$$\boxed{Q = \frac{3}{4} \pi U_0 R^2} \quad (5)$$

Verificamos unidades;

$$\ast [Q] = \frac{\text{Volumen}}{\text{Tiempo}} = \frac{L^3}{T}$$

$$\ast [Q] = \left[\frac{3}{4} \pi \right] \cdot [U_0] \cdot [R^2] = 1 \cdot [\text{Velocidad}] \cdot [\text{Area}]$$

$$[Q] = 1 \cdot \frac{L}{T} \cdot L^2 \rightarrow [Q] = \frac{L^3}{T} \quad \checkmark$$

c) Velocidad media (V_{med})?

$$V_{med} = \frac{\int_0^R u \cdot dr}{R} = \int_0^R \frac{U_0 (1 - r^6/R^6)}{R} dr$$

$$V_{med} = \frac{U_0}{R} \left[r \right]_0^R - \frac{1}{R^6} \cdot \left[\frac{r^7}{7} \right]_0^R$$

$$V_{med} = \frac{U_0}{R} \left[R - \frac{R^7}{7 \cdot R^6} \right] ; \quad V_{med} = \frac{U_0}{R} \left[R - \frac{R}{7} \right]$$

$$V_{med} = \frac{U_0}{R} \cdot R \left[1 - \frac{1}{7} \right] \rightarrow \boxed{V_{med} = U_0 \cdot \frac{6}{7}} \quad (6)$$

b) Esfuerzo de corte sobre paredes del tubo

Para fluidos newtonianos:

$$\boxed{\tau = \mu \cdot \frac{du}{dy}} \quad (7)$$

En este caso: $dy = dr$ (variables radios)

calculo $\frac{dv}{dr}$,

$$v = v_0 \left(1 - \frac{r^6}{R^6}\right) \quad (\text{ec 1})$$

$$\frac{dv}{dr} = v_0 \cdot \frac{d\left[1 - \frac{r^6}{R^6}\right]}{dr}$$

$$\frac{dv}{dr} = v_0 \left[0 - \frac{1}{R^6} \cdot 6r^5\right]$$

$$\boxed{\frac{dv}{dr} = -\frac{6v_0}{R^6} \cdot r^5} \quad , \text{ a } (\text{2}):$$

$$\boxed{\tau = M \cdot \left(-\frac{6v_0}{R^6}\right) \cdot r^5} \quad ; \text{ Para } r=R$$

$$\tau = -\frac{M \cdot 6v_0}{R^6} \cdot R^5 \rightarrow \boxed{\tau = (6 \cdot M \cdot v_0 / R)} \quad (\text{3})$$

Verifico unidades;

$$[\tau] = \frac{\text{Fuerza}}{\text{Area}} = \frac{M \cdot L}{T^2 \cdot L^2} = \frac{M}{T^2 \cdot L}$$

$$[(6 \cdot M \cdot v_0 / R)] = 1 \cdot \frac{M}{L \cdot T} \cdot \frac{L}{T} \cdot L^{-1} = \frac{M}{T^2 \cdot L}$$

} ✓

Problema 4

a) Hallar $V_{\text{media}}/V_{\text{max}}$; b) hallar \bar{y} e \bar{y}'

a) Datos: $\boxed{u(y) = U_0 \left(\frac{y}{a}\right)^{1/7}} \quad (1)$

Se define $\boxed{V_{\text{media}} = \frac{Q}{A}} \quad (2)$

Calculo Q y A por integración:
Sección transversal rectangular de
conducto.

de (2): $Q = \int V dA$, $\boxed{A = l \cdot a} \quad (3)$ — $\frac{l}{a}$: profundidad
altura

de (3): $\boxed{dA = dx dy} \quad (4)$ — $\begin{cases} x \in [0, l] \\ y \in [0, a] \end{cases}$

queda: $Q = \int_0^l \int_0^a u dy dx = \int_0^l \int_0^a U_0 \left(\frac{y}{a}\right)^{1/7} dy dx \Rightarrow$

$$\Rightarrow Q = a^{-1/7} U_0 \cdot l \int_0^a y^{1/7} dy \Rightarrow$$

$$\Rightarrow Q = a^{-1/7} U_0 \cdot l \cdot \left[\frac{7}{8} y^{8/7} \right]_0^a \Rightarrow$$

$$\Rightarrow Q = a^{-1/7} U_0 \cdot l \cdot \frac{7}{8} (a^{8/7} - 0^{8/7}) \Rightarrow$$

$$\Rightarrow \boxed{Q = a U_0 l \cdot \frac{7}{8}} ; \text{ de (3) y (2)}$$

$$V_{\text{media}} = \frac{Q}{A} = \frac{a U_0 l \cdot \frac{7}{8}}{8 \cdot l \cdot a} \Rightarrow \boxed{V_{\text{media}} = \frac{7}{8} U_0} \quad (5)$$

a) $V_{max} = V_0$, queda de (5):

$$\boxed{\frac{V_{medio}}{V_{max}} = \frac{7 \cdot V_0}{8 \cdot V_0} = \frac{7}{8}}$$

b) dado $\mu(y) = V_0 \left[\frac{y}{a} \right]^{\frac{1}{7}}$. Ver y^* tal $g' \mu(y^*) = V_m$

$$V_0 \left[\frac{y}{a} \right]^{\frac{1}{7}} = \frac{7}{8} V_0 \Rightarrow$$

$$\Rightarrow \left(\left[\frac{y}{a} \right]^{\frac{1}{7}} \right)^7 = \left(\frac{7}{8} \right)^7 \Rightarrow \boxed{\frac{y}{a} = \left(\frac{7}{8} \right)^7 \approx 0,392}$$

Problema 5

El coeficiente de Coriolis α (Factor de corrección de la energía) se define:

$$\alpha = \frac{\iint v^3 dA}{V_{mede}^3 \cdot A} \quad (1)$$

A: área V: velocidad

$$dA = db \cdot dy$$

1) Para la sección considerada: $A = b \cdot h$ (2) (rectangular)

$$V_{mede} = \frac{V_{max}}{2} \quad (3)$$

2) $V = a \cdot y + c$ (4) distribución lineal de velocidad

$$\hookrightarrow V(0) = 0 \rightarrow c = 0$$

$$\hookrightarrow V(h) = V_{max} \rightarrow V_{max} = h \cdot a \rightarrow$$

$$V(y) = \frac{V_{max} \cdot y}{h} \quad (5)$$

$$dA = db \cdot dy \quad (6)$$

queda finalmente:
$$\int v^3 dA = \int_0^b \int_0^h \left(\frac{V_m}{h}\right)^3 y^3 dy db = \frac{b \cdot V_m^3}{h^3} \cdot \frac{1}{4} y^4 \Big|_0^h$$

$$= \frac{b \cdot V_m^3}{h^3 \cdot 4} \cdot h^4 \rightarrow \frac{b \cdot V_m^3 \cdot h}{4} \quad (7)$$

En (1):
$$\alpha = \frac{\cancel{b} \cdot V_m^3 \cdot \cancel{h} \cdot 2^3}{4 \cdot V_m^3 \cdot \cancel{h} \cdot \cancel{h}} = \frac{8}{4} = 2 \rightarrow \boxed{\alpha = 2}$$

Problema 6

Hallar Fuerza pedida.

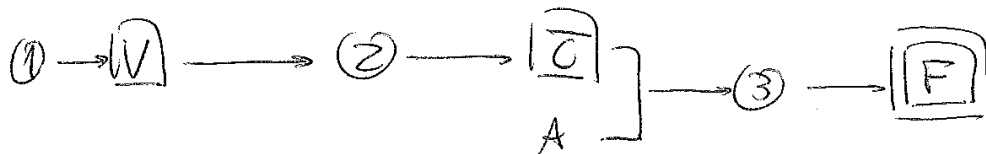
$$\boxed{V = V_m \cdot \left[1 - \frac{4}{h^2} \cdot y^2 \right]} \quad (1)$$

Para fluidos newtonianos:

$$\boxed{\tau = \mu \cdot \frac{dV}{dy}} \quad (2)$$

$$\boxed{F = \tau \cdot A} \quad (3)$$

Esquema de resolución



de (1) y (2):

$$\frac{dV}{dy} = \frac{d \left[V_m \cdot \left[1 - \frac{4}{h^2} \cdot y^2 \right] \right]}{dy} = V_m \left(\frac{d(1)}{dy} - \frac{d \left(\frac{4}{h^2} \cdot y^2 \right)}{dy} \right)$$

$$\frac{dV}{dy} = V_m \left(0 - \frac{4}{h^2} \cdot 2 \cdot y \right)$$

$$\boxed{\frac{dV}{dy} = - \frac{8 \cdot V_m \cdot y}{h^2}} \quad (4)$$

en (2):

$$\boxed{\tau = - \frac{\mu \cdot 8 \cdot V_m \cdot y}{h^2}}$$

en (3):

$$\boxed{F = - \frac{\mu \cdot 8 \cdot V_m \cdot y \cdot A}{h^2}} \quad (5)$$

En placa inferior,

$$\left[F = \frac{-8 \cdot \mu_m \cdot A \cdot \delta}{h_o^2} \right]_{\delta = -h_o} \rightarrow F = \frac{-8 \cdot \mu_m \cdot A \cdot (-h_o) \cdot \mu}{h_o^2}$$

$$\boxed{F = \frac{8 \cdot \mu_m \cdot A \cdot \mu}{h_o}} \quad \text{reemplazo valores: } \mu_{15^\circ\text{C}} = 1,139 \cdot 10^{-3} \frac{\text{kg}}{\text{m}\cdot\text{s}}$$

$h_o = 0,5 \text{ mm}$

$$F = \frac{8 \cdot 0,05 \text{ m} \cdot 0,1 \text{ m}^2 \cdot 10^3 \text{ m}^2 \cdot 10^{-3} \text{ kg} \cdot 1,139}{1 \cdot 0,5 \text{ mm} \cdot 1 \text{ m} \cdot \text{m} \cdot \text{s}}$$

$$[F] = \text{m} \cdot \frac{\text{kg}}{\text{s}^2} = \text{N} \quad \boxed{F = 2,278 \cdot 10^{-2} \text{ N}}$$

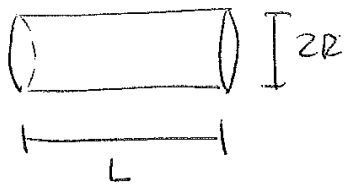
Problema 7

Hallar F

Ecuaciones

$$F = \tau \cdot A \quad (1)$$

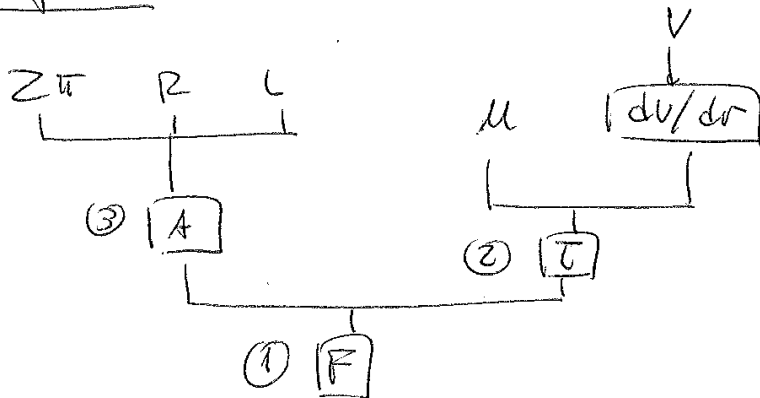
$$\tau = \mu \cdot \frac{dv}{dr} \quad (2)$$



$$A = 2\pi R \cdot L \quad (3)$$

Área por unidad de longitud "L"

Esquema:



$$\Rightarrow \frac{dv}{dr} = \frac{d(V_m (1 - r^n/R^n))}{dr} = V_m \left[\frac{d(1)}{dr} - \frac{1}{R^n} \cdot \frac{d(r^n)}{dr} \right]$$

$$\frac{dv}{dr} = V_m \left[0 - \frac{1}{R^n} \cdot n \cdot r^{n-1} \right] ; \quad \boxed{\frac{dv}{dr} = \frac{-n \cdot r^{n-1} \cdot V_m}{R^n}}$$

evaluado en $r=R$

$$\boxed{\frac{dv}{dr} = \frac{-n \cdot R^{n-1} \cdot V_m}{R^n} = \frac{-n \cdot V_m}{R}}$$

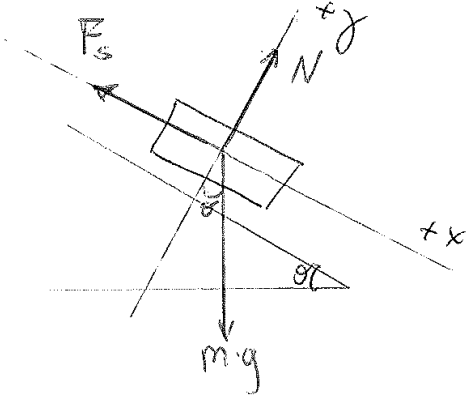
Reemplazo en (1):

$$\boxed{F = 2\pi R L \cdot \mu \cdot \left(\frac{-n \cdot V_m}{R} \right)}$$

Problema 8

Hallar velocidad final U_B

⊙ Planteo diagrama de cuerpo libre



donde F_s : Fuerza de fricción fluido

$m \cdot g$: peso del bloque

Dado que $U_B = \text{constante}$, ∴

$$\boxed{\sum F_x = 0} \text{ (1)} \quad \boxed{\sum F_y = 0} \text{ (2)}$$

Para calcular F_s :

$$\tau = \frac{F_s}{\text{Area}} = \frac{F_s}{S}$$

$$\boxed{F_s = \tau \cdot S} \text{ (3)}$$

Para fluidos newtonianos:

$$\boxed{\tau = \mu \cdot \frac{dU}{dy}} \text{ (4)}$$

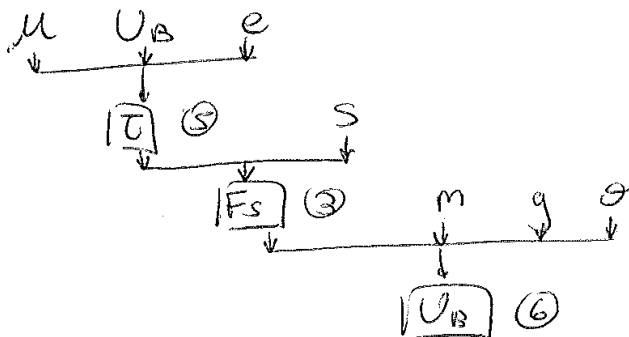
→ en perfil lineal:

$$\boxed{\tau = \frac{\mu \cdot U_B}{e}} \text{ (5)}$$

De ec. (1):

$$\boxed{\sum F_x = 0} \\ \boxed{F_s - m \cdot g \cdot \sin(\theta) = 0} \text{ (6)}$$

⇒ Esquema de resolución



de (5) y (3):

$$\boxed{F_s = \frac{\mu \cdot U_B \cdot S}{e}} \quad (3^*)$$

de (3^{ns}) y (6):

$$F_s - m \cdot g \cdot \sin(\alpha) = 0$$

$$\frac{\mu \cdot U_B \cdot S}{e} - m \cdot g \cdot \sin(\alpha) = 0$$

$$\frac{\mu \cdot U_B \cdot S}{e} = m \cdot g \cdot \sin(\alpha)$$

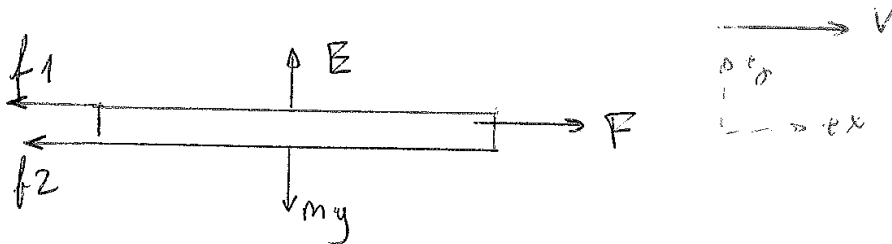
$$\frac{\mu \cdot U_B \cdot S}{e} \cdot \frac{e}{S \cdot \mu} = m \cdot g \cdot \sin(\alpha) \cdot \frac{e}{S \cdot \mu}$$

$$\boxed{U_B = \frac{m \cdot g \cdot \sin(\alpha) \cdot e}{S \cdot \mu}}$$

Problema 9

Hallar Fuerza F

⊗ Plantearmos diagrama de cuerpo libre de la placa móvil



Donde: f_1, f_2 : fricción plúvida de fluido 1 y 2

E : empuje

my = peso

F : fuerza aplicada a placa.

*) Para velocidad constante V :

$$\boxed{\sum F_x = 0} \text{ (1)} \quad \boxed{\sum F_y = 0} \text{ (2)}$$

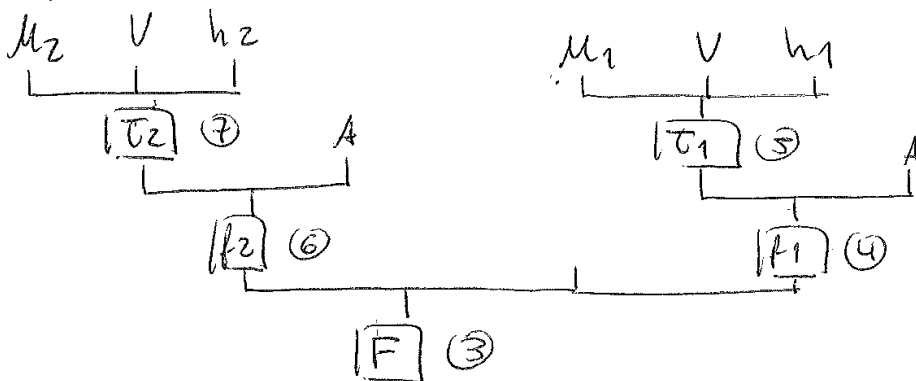
$$\sum F = F - f_1 - f_2 = 0$$

$$\boxed{F = f_1 + f_2} \text{ (3)}$$

$$\text{⊗ } \boxed{f_1 = \frac{\tau_1}{A}} \text{ (4)} \quad , \quad \text{con fluidos newtonianos: } \boxed{\tau_1 = \frac{\mu_1 \cdot V}{h_1}} \text{ (5)}$$

$$\text{⊗ } \boxed{f_2 = \frac{\tau_2}{A}} \text{ (6)} \quad , \quad \boxed{\tau_2 = \frac{\mu_2 \cdot V}{h_2}} \text{ (7)}$$

Esquema de resolución



de (7) en (6) y (5) y (4)

$$f_1 = \frac{m_1 \cdot v \cdot A}{h_1}$$

$$f_2 = \frac{m_2 \cdot v \cdot A}{h_2}$$

de (5): $F = \frac{m_1 \cdot v \cdot A}{h_1} + \frac{m_2 \cdot v \cdot A}{h_2}$

$$F = v \cdot A \left(\frac{m_1}{h_1} + \frac{m_2}{h_2} \right) \quad (8)$$

Verifiquemos unidades:

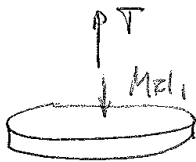
$$[F] = \frac{M \cdot L}{T^2}$$

$$[F] = [v] \cdot [A] \cdot \left[\frac{M}{h} \right] = \frac{L}{T} \cdot L^2 \cdot \frac{M}{L \cdot T} \cdot \frac{1}{L} \quad \therefore [F] = \frac{M \cdot L}{T^2}$$

} ✓

Problema 10

Hallar ω_2 para el disco 1, $\omega_1 = \text{cte}$, i.e. $\boxed{\sum M = 0}$ (1)

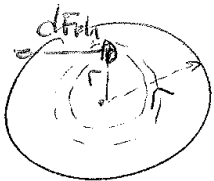


M_{FL1} = momento aplicado por fricción fluido en disco 1.

de (1): $\boxed{T - M_{FL1} = 0}$ (2)

$M_{FL1} = F_{FL1} \cdot R$ \rightarrow resolvemos con integral:

$\boxed{dM_{FL1} = r \cdot dF_{FL1}}$ (3)



$\boxed{dF_{FL1} = \tau \cdot dA}$ (4)

Tomamos áreas de anillo:

$A = \pi \cdot r^2 \rightarrow \boxed{dA = 2\pi r dr}$ (5)

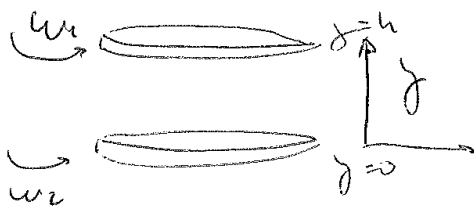
Suponemos fluido newtoniano:

$\boxed{\tau = \mu \cdot \frac{dv}{dz}}$ (6)

Siendo y ; coordenada vertical normal a disco 1 y 2.

Por perfil lineal, $\boxed{V = a \cdot y + b}$ (7), $a, b \in \mathbb{R}$

El fluido tiene la misma velocidad que los puntos de contacto con los discos que los propios discos.



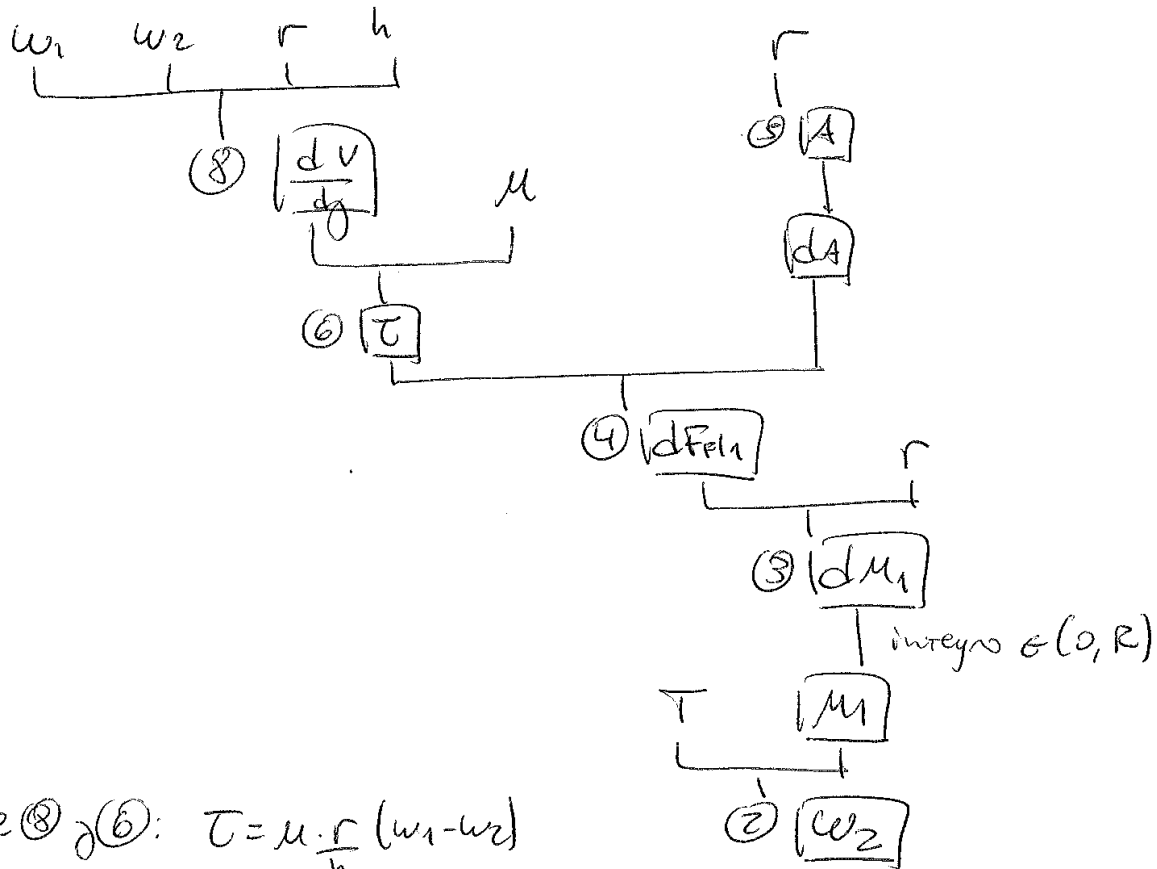
$V(0) = \omega_2 \cdot r$
 $V(h) = \omega_1 \cdot r$

reemplazo el valor a y b

$\boxed{V = \frac{r}{h} (\omega_1 - \omega_2) \cdot y + \omega_2 \cdot r}$ (7)

$$\left| \frac{dv}{dy} = \frac{r}{h} (\omega_1 - \omega_2) \right| \quad (8)$$

Esquema de resolución



de (8) y (6): $T = \mu \cdot \frac{r}{h} (\omega_1 - \omega_2)$

de (6) y (5) e (4): $dF_{Fl1} = \frac{\mu \cdot r}{h} (\omega_1 - \omega_2) \cdot 2\pi r dr$

de (3): $\int_0^R dM_1 = \int_0^R \frac{\mu}{h} \cdot 2\pi (\omega_1 - \omega_2) \cdot r^3 dr$

$$M_1 = \frac{\mu \cdot 2\pi (\omega_1 - \omega_2) \cdot R^4}{4h}, \quad \text{con } R = \phi/2$$

$$M_1 = \frac{\mu \cdot 2\pi (\omega_1 - \omega_2) \cdot (\phi/2)^4}{4h} = T$$

de (2):

$$\omega_1 - \omega_2 = \frac{T \cdot 4 \cdot h}{2\pi \mu (\phi/2)^4}$$

$$\left| \omega_2 = \omega_1 - \frac{T \cdot 4 \cdot h}{2\pi \mu (\phi/2)^4} \right|$$